

Ceiling diffusers A1 ÷ A4



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Ceiling diffusers A1+A4

Ceiling diffusers A1 ÷ A4 have fixed blades configured for air supply into 1 ÷ 4 directions. They are suitable for air supply in air-conditioning and ventilation systems and ceiling or air duct installation.

A1÷A4 🗾

Ceiling diffusers A1 \div A4 can be manufactured from anodized aluminium or aluminium painted in RAL color :

A1 ÷ A4... : Blades & frame from anodized aluminium or aluminium painted in RAL color.

They can be installed in spaces with height up to 4 m and they are ideal for systems with variable flow rate as the configuration of the blades achieves steady stream morphology at high velocities, eliminating the risk of flow detachment from the ceiling in case of low air supply. The ability to achieve airflow with high velocities makes the grilles $A1 \div A4$ suitable for installation in spaces with large temperature difference between the air within the space and the supplied air.



CEILING DIFFUSERS A1 + A4 SIZE SELECTION

The selection of ceiling diffusers $A1 \div A4$ will be made using the following diagrams and in accordance with the guideline CR 1752:1998 (Ventilation for buildings - Design criteria for the indoor environment).

The technical specifications for ceiling diffusers A1 ÷ A4 are the following :

Diffuser lenght	L	[mm]
Diffuser height	Н	[mm]
Diffuser surface factor	Af	
Pressure drop inside the diffuser	ΔΡ	[Pa]
Maximum air velocity inside the diffuser	U	[m/s]
Noise level	ΘČ	dB[A]
Temperature difference Supply / Room	ΔΤ	°C
Horizontal stream range	X	[m]
Horizontal stream vertical drop	Y	[m]
Horizontal stream velocity at distance X	υŢ	[m/s]
Horizontal air-stream temperature	T,	°C
Distance between diffuser and point of stream collision	A,	[m]

Upon request ceiling diffusers **A1** \div **A4** may have volume damper, air supply equalizing grid, can be manufactured as accessible ceiling diffusers with removable blade core or can be installed in a false ceiling plate 595 x 595 mm.

With equalizing grid



With volume damper



With equalizing grid and volume damper





- 5,5

CEILING DIFFUSERS A1 ÷ A4 TYPES

A1	From aluminium. Fixed blades configured for air supply into 1 direction.
A2	From aluminium. Fixed blades configured for air supply into 2 directions.
A3	From aluminium. Fixed blades configured for air supply into 3 directions.
A4	From aluminium. Fixed blades configured for air supply into 4 directions.
A1 ÷ A4 +D	From aluminium. Diffusers A1 ÷ A4 with volume damper.
A1 ÷ A4 +E	From aluminium. Diffusers A1 ÷ A4 with equalizing grid.
A1 ÷ A4 +D+E	From aluminium. Diffusers A1 ÷ A4 with volume damper and equalizing grid

CEILING DIFFUSERS A1 + A4 INSTALLATION

1. Visible installation with screws

For easy, fast and safe installation. The number of screws needed is proportional to the size of the diffuser. In case the diffuser is very large, can be provided fragmented according to the requirements.

2. Concealed placement with internal screws, on the side of the diffuser

For an aesthetically better result and a secure installation. The diffuser is held in the hole with internal screws on the sides of the diffuser. The screws are accessible through the opening face of the diffuser.



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CEILING DIFFUSERS A1 + A4 PLACEMENT









PRESSURE DROP & NOISE LEVEL CALCULATION

Selection example 1:

Which is the pressure drop and the produced noise level in a diffuser A2 (2 directions) 300 x 300 mm, if the air flow is 400 m³/h;

From the surface factor Af selection table, we establish that according to the diffuser's dimensions the surface factor Af is equal to 90. From diagram 1.1, for air flow of 400 m³/h and surface factor 90, we estimate that the maximum air velocity inside the diffuser is 2,6 m/s, while from diagram 1.2 we estimate that the produced noise is 27,4 dB(A). Simiral, from diagram 3 (A2), for air flow of 400 m³/h and surface factor 90, we estimate that the pressure drop is equal to 29,3 Pa.

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NOTE :	SURFACE FACTOR SELECTION TABLE									
		150	230	300	380	450	530	610	700	800
The produced noise level and the velocity inside the diffuser are calculated using the following diagrams 1.2 and 1.1 respectively, which are the same for all types of A1 \div A4 diffusers. The calculation of the pressure drop, however, depends upon the type of the diffuser (A1, A2, A3, A4) and is done using the following diagrams 2 \div 5 respectively (page 6 \div 7).	150	22,5	34,5	45,0	57,0	67,5	79,5	91,5	105,0	120,0
	230	34,5	52,9	69,0 🔰	87,4	103,5	121,9	140,3	161,0	184,0
	300	45,0	_ 69,0	90,0	114,0	135,0	159,0	183,0	210,0	240,0
	380	57,0	87,4	114,0	144,4	171,0	201,4	231,8	266,0	304,0
	450	67,5	103,5	135,0	171,0	202,5	238,5	274,5	315,0	360,0
	530	79,5	121,9	159,0	201,4	238,5	280,9	323,3	371,0	424,0
	610	91,5	140,3	183,0	231,8	274,5	323,3	372,1	427,0	488,0
	700	105,0	161,0	210,0	266,0	315,0	371,0	427,0	490,0	560,0
	800	120,0	184,0	240,0	304,0	360,0	424,0	488,0	560,0	640,0

The diagrams are an approximate selection method for A1 ÷ A4 diffusers. For more precise calculation, please use the *AIRTECHNIC* air grilles calculation software or contact us.





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DIFFUSER'S PRESSURE DROP & NOISE LEVEL

Calculation example 1 :

Pressure drop and noise level calculation in a diffuser A3 + Damper with blade angle of 15°

We have a diffuser A3 + Damper with dimensions **450 x 450** and air flow of 800 m³/h. A diffuser A3 with dimensions **450 x 450**, has, according to diagram 4 (page 7), for air flow equal to 800 m³/h, pressure drop equal to 11,8 Pa and according to diagram 1 (page 5), produces noise level of 24,2 dB. A damper with dimensions **450 x 450** has, according it's respective selection diagrams, for blade angle of 15° and air flow of 800 m³/h, pressure drop equal to 5,6 Pa and produces noise equal to 10,2 dB.

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The total pressure drop inside the diffuser A3 + Damper with dimensions **450 x 450** is the algebraic sum of the pressure drop inside the diffuser and the pressure drop inside the damper: $\Delta p_{A3} + \Delta p_{Damper} = 11.8 + 5.6 = 17.4$ Pa.

The total noise level is calculated by using the following equation: $L_{tot} = L_{A3} \oplus L_{Damper} = Lmax + C(\Delta L)$. The difference between the noise levels of the 2 independent sound sources (the diffuser A3 and the damper) is $\Delta L = 14$ dB. Therefore from the following diagram we determine that for $\Delta L = 14$ dB the correction factor $C(\Delta L)$ is equal to 0,1. So, the total noise level is $L_{tot} = Lmax + C(\Delta L) = 24,2 + 0,1 = 24,3$ dB.

Calculation example 2 :

Pressure drop and noise level calculation in a diffuser A1 + Damper with blade angle of 45°

We have a diffuser A1 + Damper with dimensions **450 x 450** and air flow of 1.000 m^3 /h. The diffuser A1 with dimensions **450 x 450** has, according to diagram 2 (page 6), for air flow equal to 1.000 m^3 /h, pressure drop equal to 17,8 Pa and according to diagram 1 (page 5), produces noise level of 30,2 dB. A damper with dimensions **450 x 450** has, according it's respective selection diagrams, for blade angle of 45° and air flow of 1000 m³/h, pressure drop equal to 59,4 Pa and produces noise equal to 40,5 dB.

The total pressure drop inside the diffuser A1 + Damper with dimensions **450 x 450** is the algebraic sum of the pressure drop inside the diffuser and the pressure drop inside the damper: $\Delta p_{A1} + \Delta p_{Damper} = 17,8 + 59,4 = 77,2$ Pa.

The total noise level is calculated by using the following equation: $L_{tot} = L_{A1} \oplus L_{Damper} = Lmax + C(\Delta L)$. The difference between the noise levels of the 2 independent sound sources (the diffuser A1 and the damper) is $\Delta L = 10,3 \text{ dB}$. From the following diagram we determine that for $\Delta L = 10,3 \text{ dB}$ the correction factor $C(\Delta L)$ is equal to 0,38. So, the total noise level is $L_{tot} = Lmax + C(\Delta L) = 40,5 + 0,38 = 40,88 \text{ dB}$.



CALCULATING THE TOTAL NOISE LEVEL BETWEEN 2 INDEPENDENT SOUND SOURCES

Since noise in [dB] is a quantity that is defined in logarithmic scale, when we have 2 (or more) independent sound sources, the total noise is not calculated by the algebraic sum of the 2 sources. The "sum" of 2 sound sources L1, L2 is symbolized by the internationaly defined symbol \oplus and is calculated by using the following equation :

 $L_{tot} = L1 \oplus L2 = 10 \times \log(10^{0.1 \times L1} + 10^{0.1 \times L2})$

Because of the previous equation requiring some complex calculations, we can define the sum of 2 sound sources with sufficient accuracy using the following approximate equation :

 $L_{tot} = L1 \oplus L2 = Lmax + C(\Delta L),$

where Lmax is the largest noise level between L1 and L2 and C(Δ L) a correction factor (in dB) which depends on the difference Δ L = | L2 - L1 | and is calculated by using the following diagram.



RANGE CALCULATION - A1

Selection example 2 :

Which is the total range of a diffuser A1 with dimensions 300 x 300, if the air flow is 300 m³/h, we have collision between the air stream of this diffuser and the air stream of another's, at a distance of 1 m from each diffuser and the stream velocity at total range is 0,2 m/s?

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From the surface factor Af selection table (page 5) we establish that, for a diffuser with dimensions **300 x 300**, the surface factor Af is equal to 90. Therefore, from diagram 6.1, for air flow of 300 m³/h and surface factor Af = 90, we determine the factor $S_1 = 1,74$. We continue to diagram 6.2, where, for factor $S_1 = 1,74$ and stream velocity at total range equal to 0,2 m/s, we determine the factor $S_2 = 3,4$. From diagram 6.3, for factor $S_2 = 3,4$ and the curve for collision distance equal to $A_s = 1$ m, we determine the factor $S_3 = 3,4$. Finally, from diagram 6.4 for factor $S_3 = 3,4$ and the curve for collision between streams, we determine that the stream vertical drop Y_0 is equal to 3,4 m. The total range is calculated by the equation $X_0 = A_s + Y_0 = 1 + 3,4 = 4,4$ m.



RANGE CALCULATION - A2

Selection example 3 :

Which is the total range, per direction, of a diffuser A2 with dimensions 230 x 230, if the air flow is 200 m³/h, we have collision between the stream of the diffuser and the wall located 1 m away from the diffuser and the stream velocity at total range is 0,2 m/s?

A1÷A4 🔟

From the surface factor Af selection table (page 5) we establish that, for a diffuser with dimensions **230 x 230**, the surface factor Af is equal to 52,9. Therefore, from diagram 7.1, for air flow of 200 m³/h and surface factor 52,9, we determine the factor $S_1 = 0,89$. We continue to diagram 7.2, where, for factor $S_1 = 0,89$ and stream velocity at total range equal to 0,2 m/s, we determine the factor $S_2 = 1,86$. From diagram 7.3, for factor $S_2 = 1,86$ and the curve for collision distance equal to $A_s = 1$ m, we determine the factor $S_3 = 1,86$. Finally, from diagram 7.4 for factor $S_3 = 1,86$ and the curve for collision between the stream of the diffuser and the wall, we determine that the stream vertical drop Y_0 is equal to 3,02 m. The total range **per direction** is calculated by the equation $X_a = A_s + Y_a = 1 + 3,02 = 4,02$ m.



RANGE CALCULATION - A3 / TRIANGULAR SURFACE

Selection example 4 :

Which is the total range of the stream discharged from the triangular surface af a diffuser A3 with dimensions 300 x 230 if the total air flow is 300 m³/h, we have collision between the air stream of this diffuser and the air stream of another's, at a distance of 1 m from each diffuser and the stream velocity at total range is 0,2 m/s?

A1÷A4

From the surface factor Af selection table (page 5) we establish that, for a diffuser with dimensions **300 x 230** the surface factor Af is equal to 69. From diagram 8.1 for air flow of 300 m³/h and surface factor 69, we determine the factor $S_1 = 0.83$. We continue to diagram 8.2, where, for factor $S_1 = 0.83$ and stream velocity at total range equal to 0.2 m/s, we determine the factor $S_2 = 1.75$. From diagram 8.3, for factor $S_2 = 1.75$ and the curve for collision distance equal to $A_s = 1$ m, we determine the factor $S_3 = 1.74$. Finally, from diagram 8.4 for factor $S_3 = 1.74$ and the curve for collision between streams, we determine that the stream vertical drop Y₀ is equal to 1.75 m. The total range is calculated by the equation $X_0 = A_s + Y_0 = 1 + 1.75 = 2.75$ m.



RANGE CALCULATION - A3 / TRAPEZOIDAL SURFACE

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Selection example 5 :

Which is the total range of the stream discharged from the trapezoidal surface af a diffuser A3 with dimensions 300 x 230, of selection example 4 on page 11, if we have collision between the air stream of the diffuser and a wall located 1 m away from the diffuser and the stream velocity at total range is 0,2 m/s?

From diagram 9.1 for total air flow of 300 m³/h and surface factor equal to 69, we determine the factor $S_1 = 1$. We continue to diagram 9.2, where, for factor $S_1 = 1$ and stream velocity at total range equal to 0,2 m/s, we determine the factor $S_2 = 2,01$. From diagram 9.3, for factor $S_2 = 2,01$ and the curve for collision distance equal to $A_s = 1$ m, we determine the factor $S_3 = 2,01$. Finally, from diagram 9.4 for factor $S_3 = 2,01$ and the curve for collision between the stream and the wall, we determine that the stream vertical drop Y₀ to is equal to 3,4 m. The total range of the stream discharged from each trapezoidal surface is calculated by the equation $X_0 = A_s + Y_0 = 1 + 3,4 = 4,4$ m.



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RANGE CALCULATION - A4 / TRIANGULAR SURFACE



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RANGE CALCULATION - A4 / TRAPEZOIDAL SURFACE

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A1 ÷ A4 - ORDER

For the proper order of ceiling diffusers A1 ÷ A4 please use the following code :



Examples

A2 300 x 300 + E | 9010 =

Ceiling diffuser A2 (air supply into 2 directions) **300 mm** in length and height, with blades and frame from aluminium, powder painted in RAL 9010 and equalizing grid.

A3 380 x 450 + D =

Ceiling diffuser A3 (air supply into 3 directions), 380 mm in length, 450 mm in height, with blades and frame from anodized aluminium and volume damper.

A4 380 x 450 + E + D =

Ceiling diffuser A4 (air supply into 4 directions), **380 mm** in length, **450 mm** in height, with blades and frame from anodized aluminium, with equalizing grid and volume damper.

SPECIFICATION

Ceiling diffuser, 1 / 2 / 3 / 4 directions, A1 / A2 / A3 / A4

Ceiling diffuser, indicative type A1 / A2 / A3 / A4 of AIRTECHNIC, manufactured of anodized aluminum / aluminum painted in RAL... color, with fixed blades configured for air supply into 1 direction (A1) / 2 directions (A2) / 3 directions (A3) / 4 directions (A4). The manufacturer will have performed measurements of the technical characteristics of the diffuser, in an independent laboratory according to the standard ELOT EN 12238: 2002. It will have a volume damper [D] / equalizing grid [E]. It will be suitable for ceiling or air duct placement, for air supply and visible installation with screws / concealed placement with internal screws, on the side of the diffuser. It will be possible to be manufactured as accessible ceiling diffuser with removable blade core. The factory will be certified according to ISO 9001:2015 (Quality Management Systems) and according to ISO 14001:2015 (Environmental Management Systems).

It will be manufactured by AIRTECHNIC type A1 / A1 +D, +E.

- It will be manufactured by AIRTECHNIC type A2 / A2 +D, +E.
- It will be manufactured by **AIRTECHNIC** type **A3** / **A3** +**D**, +**E**. It will be manufactured by **AIRTECHNIC** type **A4** / **A4** +**D**, +**E**.
- It will be manufactured by AIRTECHNIC type A47 A4 +D, +E









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